















- system fitted with helical twisted tapes. Sol Energy 83: 1943–52. <https://doi.org/10.1016/j.solener.2009.07.006>
- [22] Hasan MA, Sumathy K. (2009). Study on the potential use of helical swirl generators in enhancing the thermal performance of solar air heaters. Int J Ambient Energy 30: 207–16. <https://doi.org/10.1080/01430750.2009.9675098>
- [23] Watmuff JH, Charters WWS, Proctor D. (1977). Solar and wind induced external coefficients - Solar collectors. Coop Mediterr Pour l'Energie Solaire, Rev Int d'Heliotechnique, 2nd Quarter, 56.
- [24] Heaton HS, Reynolds WC, Kays WM. (1964). Heat transfer in annular passages. Simultaneous development of velocity and temperature fields in laminar flow. Int J Heat Mass Transf 7: 763–81. [https://doi.org/http://dx.doi.org/10.1016/0017-9310\(64\)90006-7](https://doi.org/http://dx.doi.org/10.1016/0017-9310(64)90006-7)
- [25] Hong SW, Bergles A. (1976). Augmentation of laminar flow heat transfer in tubes by leans of twisted-tape inserts. J Heat Transfer 98: 251–6.
- [26] Bas H, Ozceyhan V. (2012). Heat transfer enhancement in a tube with twisted tape inserts placed separately from the tube wall. Exp Therm Fluid Sci 41: 51–8. <https://doi.org/10.1016/j.expthermflusci.2012.03.008>
- [27] Date AW, Saha SK. (1990). Numerical prediction of laminar flow and heat transfer characteristics in a tube fitted with regularly spaced twisted-tape elements. Int J Heat Fluid Flow 11: 346–54. [https://doi.org/10.1016/0142-727X\(90\)90058-J](https://doi.org/10.1016/0142-727X(90)90058-J)
- [28] Esen H. (2008). Experimental energy and exergy analysis of a double-flow solar air heater having different obstacles on absorber plates. Build Environ 43: 1046–54. <https://doi.org/10.1016/j.buildenv.2007.02.016>
- [29] Kalogirou SA, Karellas S, Braimakis K, Stanciu C, Badescu V. (2016). Exergy analysis of solar thermal collectors and processes. Prog Energy Combust Sci 56: 106–37. <https://doi.org/10.1016/j.pecs.2016.05.002>
- [30] Gupta MK, Kaushik SC. (2009). Performance evaluation of solar air heater for various artificial roughness geometries based on energy, effective and exergy efficiencies. Renew Energy 34: 465–76. <https://doi.org/10.1016/j.renene.2008.06.001>

## NOMENCLATURE

$A_c, A_p$	area of collector and absorber plate respectively ( $m^2$ )
$C$	conversion factor
$c_p$	specific heat of air (J/kg-K)
$D_h$	hydraulic diameter (m)
$Ex_{i,f}$	exergy at inlet of fluid
$Ex_{o,f}$	exergy at outlet of fluid
$Ex_{r,f}$	exergy radiated from the sun (W)
$F_p$	fin pitch (distance between two fins, m)
$f$	fanning friction factor
$H$	duct height (m)

$H_f$	fins height (m)
$h_w$	heat transfer coefficient due to wind flowing over the glass cover ( $W/m^2K$ )
$h_{r,ga}, h_{r,pg}, h_{r,pb}$	radiative heat transfer coefficient between glass cover and ambient, absorber plate and glass cover, and absorber plate and bottom plate respectively ( $W/m^2K$ )
$h_{c,pg}, h_{c,pf}, h_{c,fb}$	convective heat transfer coefficient between absorber plate and glass cover, absorber plate and air stream, air stream and bottom plate respectively ( $W/m^2K$ )
$I$	radiation intensity ( $W/m^2$ )
$k_{air}, k_{ins}, k_{fn}$	thermal conductivity of air, insulation and fins respectively ( $W/mK$ )
$L$	distance between glass cover and absorber plate (m)
$L_1$	length of the collector (m)
$L_2$	width of the collector (m)
$\dot{m}$	mass flow rate of air (kg/s)
$N$	number of fins
$Nu_{pg}$	Nusselt number between absorber plate and glass cover.
$Nu_{pb}$	Nusselt number between absorber plate and bottom plate.
$p$	pitch for $180^\circ$ rotation of twisted tape (m)
$Q_u$	useful heat gain (W)
$Re$	Reynolds number
$T_i, T_o$	inlet and outlet temperature of air (K)
$T_a$	ambient temperature (K)
$T_g, T_p, T_b$	average temperature of glass cover, absorber plate, bottom plate and fluid respectively (K)
$T_f$	thickness of fins and insulation respectively (m)
$t_f, t_{ins}$	thickness of fins and insulation respectively (m)
$U_b$	bottom heat loss coefficient ( $W/m^2K$ )
$V_w$	wind velocity (m/s)
$w$	width of twisted tape which equals to fin pitch
$Y$	twist ratio = $p/w$ , dimensionless
$\alpha_p, \alpha_g$	absorptivity of absorber plate and glass cover respectively
$\varepsilon_p, \varepsilon_b, \varepsilon_g$	emissivity of absorber plate, bottom plate and glass cover respectively
$\sigma$	Stefan's constant ( $5.67 \times 10^{-8} Wm^{-2}K^{-4}$ )
$\Delta p$	pressure drop ( $N/m^2$ )
$\rho$	density of air ( $kg/m^3$ )
$\eta_f$	fins efficiency
$\eta_{th}, \eta_{eff}$	Thermal, effective or thermohydraulic and exergy efficiency respectively
$\eta_{II}$	
.	